

High flux heat transfer in a target environment

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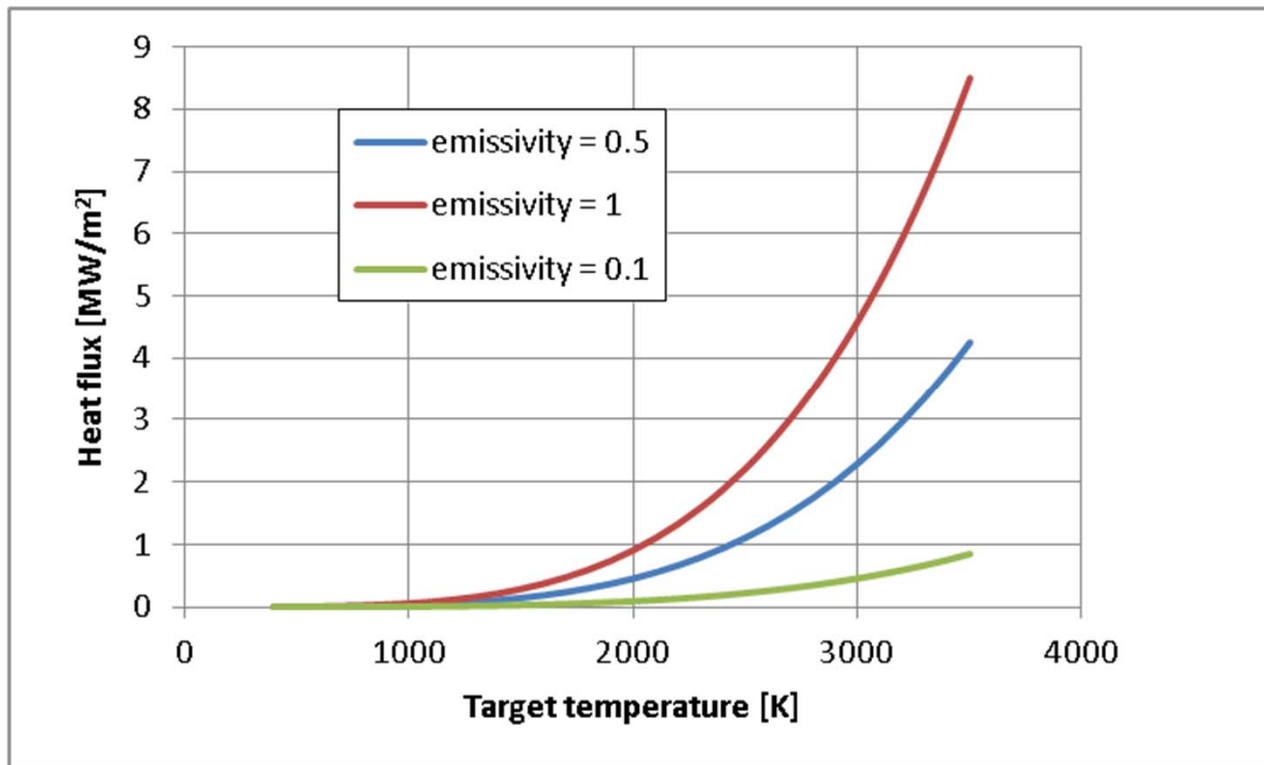
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Radiation cooling

$$Q \left[\frac{W}{m^2} \right] = \sigma \varepsilon (T_H^4 - T_C^4)$$



High temperatures require refractory metals and also good vacuum quality to avoid target loss through oxidation and evaporation cycles

Forced Convection

Consider turbulent heat transfer in a
1.5mm diameter pipe –
Dittus Boelter correlation

$$Nu_D = 0.023Re_D^{4/5} Pr^n$$

$N = 0.4$ for fluid being heated

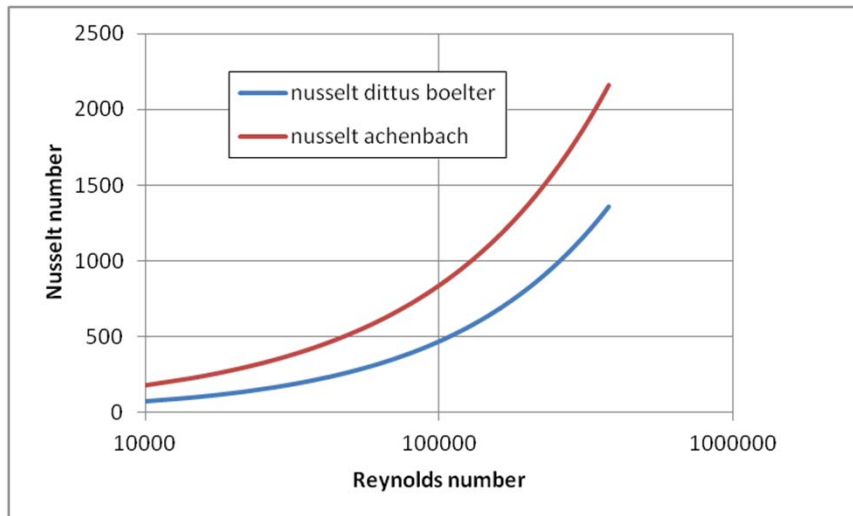
Valid for: $Re_D \gtrsim 10\,000$

$$0.6 \leq Pr \leq 160$$

	velocity [m/s] (Mach=0.3 for gases)	Pr	Re	Nu	heat transfer coefficient [W/m ² K]	allowable temp rise [K]	heat flux [MW/m ²]
air at 300K 1bar	100	0.72	11114	35	557	500	0.22
air at 300K at 10bar	100	0.73	111958	222	3558	500	1.4
helium at 300K at 1bar	300	0.67	4235	15	1516	500	0.6
helium at 300K at 10bar	300	0.67	42112	98	9520	500	3.74
helium at 1023K at 10 bar	560	0.68	8400	27	6514	500	2.56
water at 300K and 5bar	5	6.13	8823	68	26344	100	2.6
water at 300K and 5bar	10	6.13	17647	119	45868	100	4.6
water at 300K and 5bar	15 (erosion limited?)	6.13	26470	164	63444	100	6.3

$$Nu = \frac{hd}{k_g} = [(1.18Re^{0.58})^4 + (0.23Re^{0.75})^4]^{0.25}$$

Achenbach correlation for heat transfer in a packed bed of spheres



Max power density for a sphere

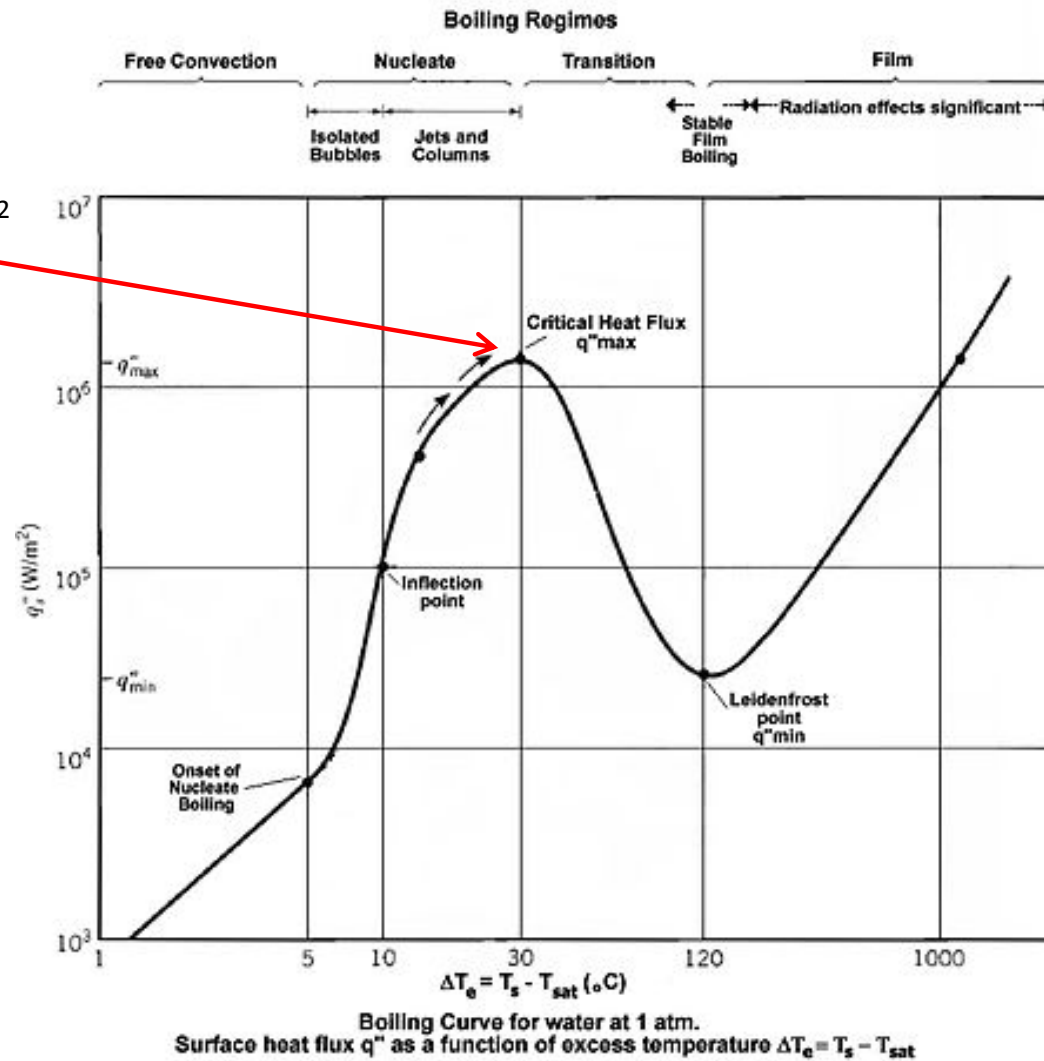
$$\frac{Q}{V} = \frac{hA\Delta T}{V}$$

for a sphere $A = 4\pi r^2$ and $V = \frac{4}{3}\pi r^3$ so

$$\frac{Q}{V} = \frac{3h\Delta T}{r} = \frac{3 * 5000 \left[\frac{W}{m^2 K} \right] * 500 [K]}{0.003 [m]} = 2.5 \left[\frac{GW}{m^3} \right] = 2.5 \left[\frac{MW}{l} \right]$$

Nucleate Boiling

Vapour bubbles forming at nucleation sites and separating from the heated surface thus enhances mixing and heat transfer

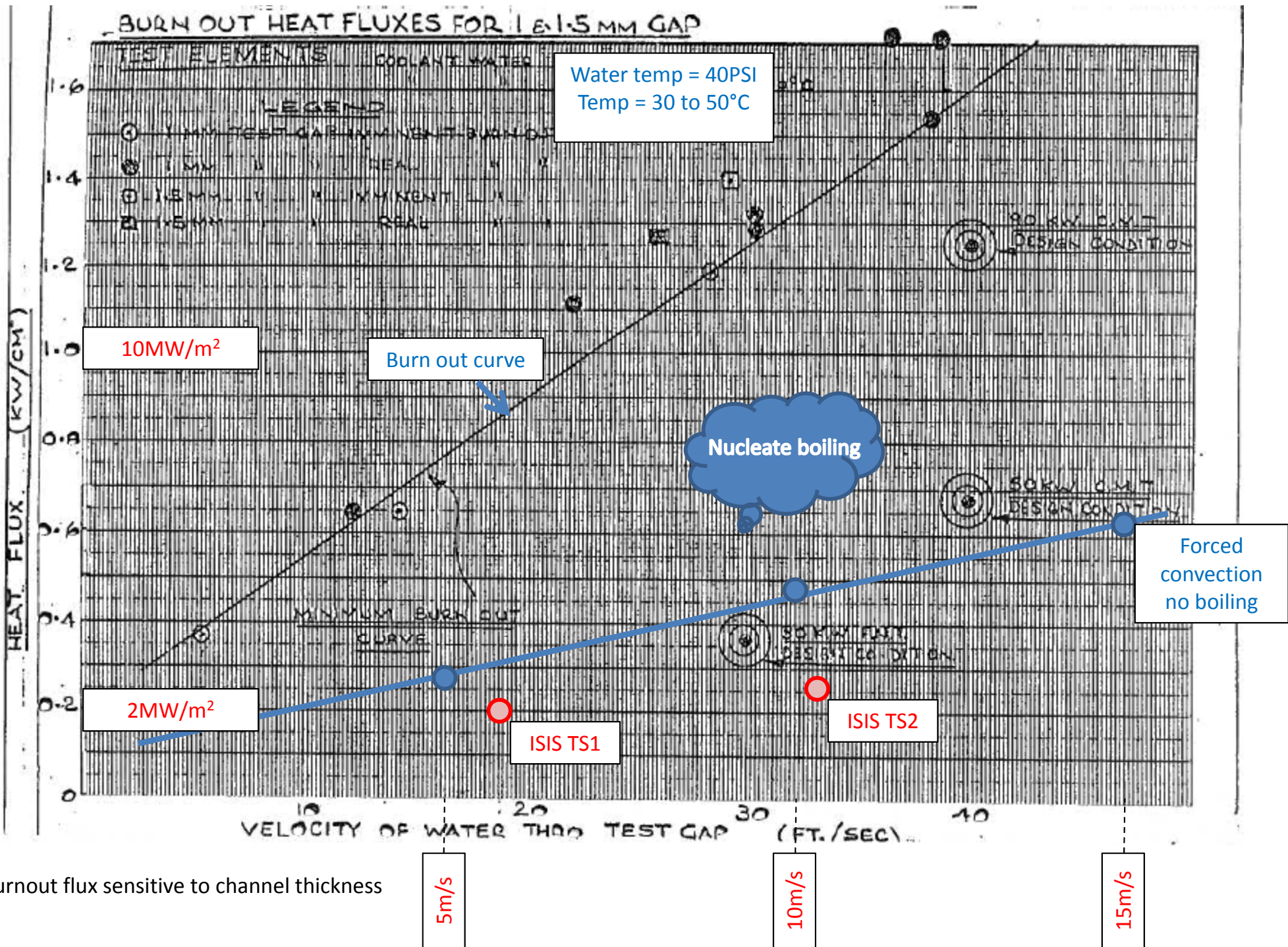


Critical heat flux $>1\text{MW}/\text{m}^2$

Heat transfer driven by temperature difference alone, i.e. Plate above boiling temperature of water and no forced convection

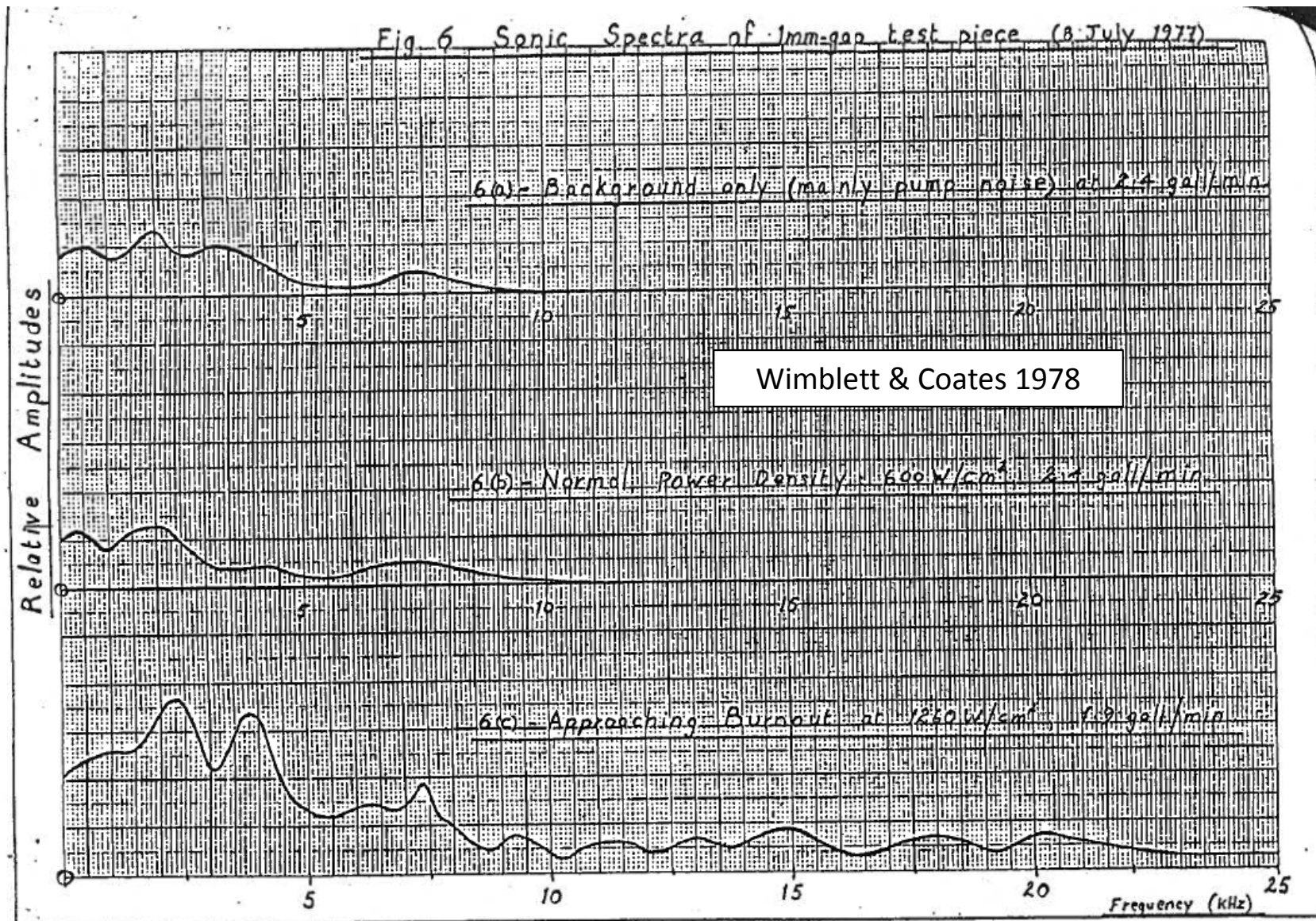
Critical heat flux

forced convection water flow (original graph Wimblett)



Burnout flux sensitive to channel thickness

Acoustic transducer used to detect burnout

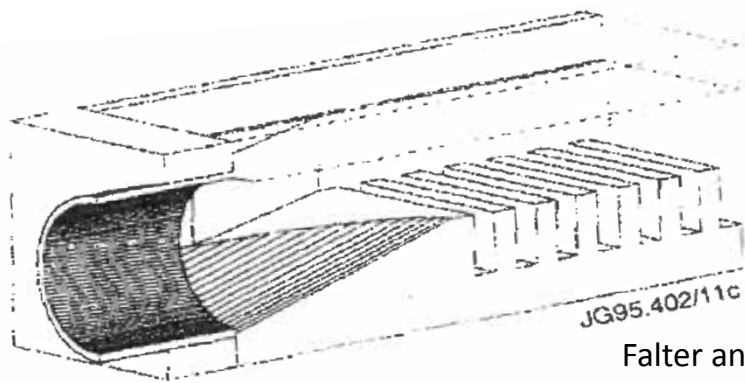


Maximum heat flux could be achieved by monitoring for burnout
Heat flux may be limited by erosion due to high water velocities

Other ideas

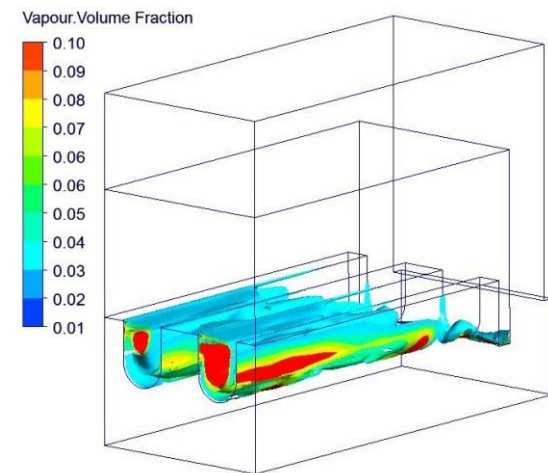
Hypervapotrons

- Water cooled finned heat exchangers developed to cope with the high heat fluxes present in experimental fusion devices and ancillary systems.
- Water flow, heat load and channel width tuned to generate a repetitive cycle that moves steam out into the sub cooled bulk flow.
- Typically, these can sustain power densities of up to 20-30 megawatts/m² in steady-state, using water at flow velocities < 10 m/s and operating pressures < 10 bar.



Falter and Thompson
Jet

Fig.2a: Hypervapotron beam stopping elements as used in the JET Test Bed beam bump.

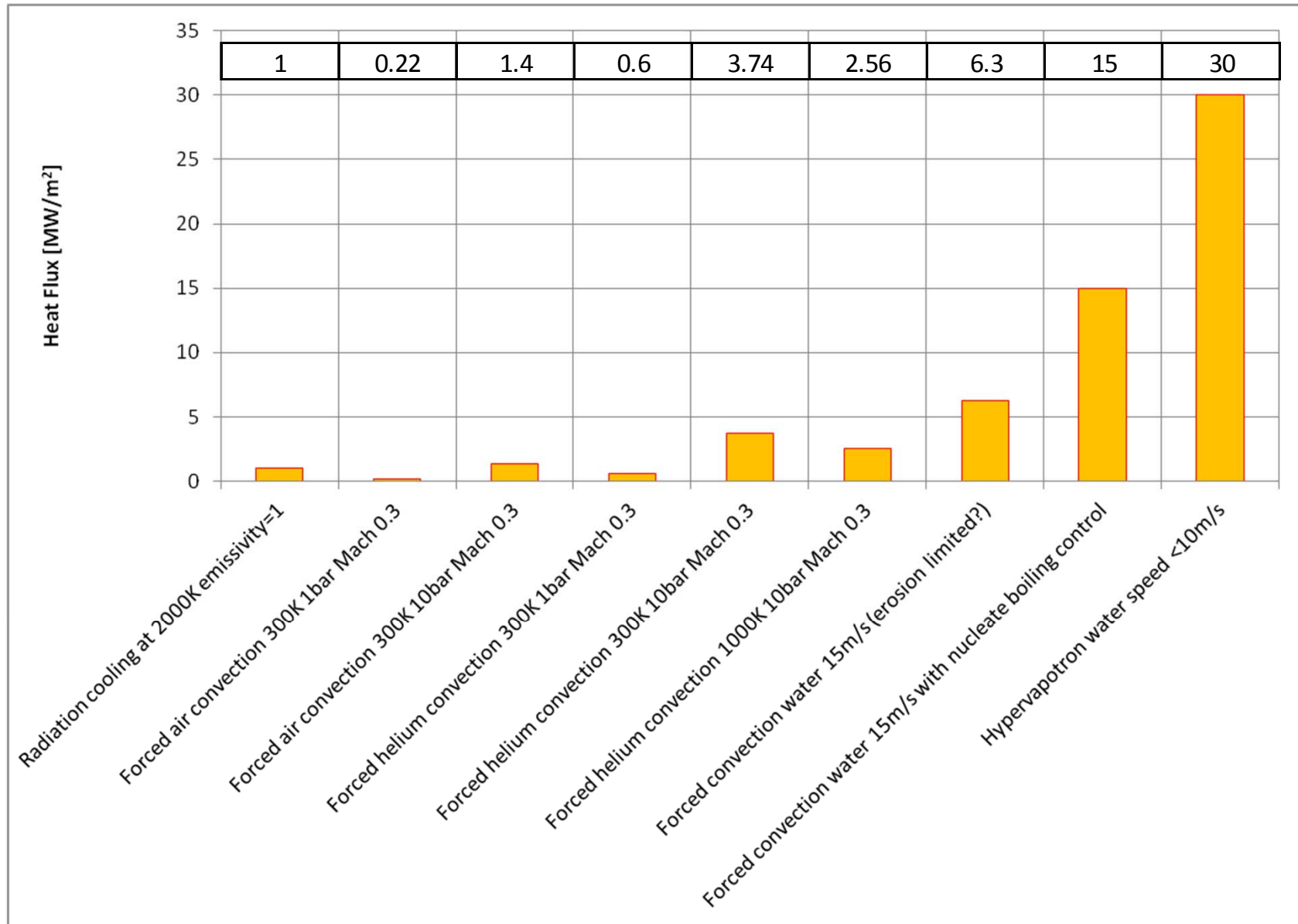


Nanofluids

- Water-based nanofluids (suspensions of 0.001-10% nanoparticles, <100nm) have the potential to deliver much improved cooling while retaining the advantages of water.
- 10-14% increase in convective/conductive heat transfer and 100-200% increase in critical heat flux have been reported.
- Long-term stability of nanofluids, the deposition of particles, and their effect on erosion are not well understood.

S. K. Das et al., Nanofluids, First ed., John Wiley & Sons, 2007

Summary



The Calculation of Critical Heat Flux in Forced Convection Boiling

[P. B. Whalley](#), [G. F. Hewitt](#), [P. Hutchinson](#)

[0 Reviews](#)

Atomic Energy Research Establishment, 1973 - 17 pages

[International Journal of Heat and Mass Transfer](#)

[Volume 30, Issue 11](#), November 1987, Pages 2261–
2269

**Critical heat flux of forced convective boiling in
uniformly heated vertical tubes with special
reference to very large length-to-diameter ratios**

[Journals](#) > [Heat Transfer Research](#) > [Volume
33, 2002 Issue 5&6](#) > **Calculation of Critical
Heat Flux in Natural and Forced Convection
Boiling**