

# **Design Principles for High Power Targets**

**4th HPTW in Malmö** 

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**□** 1. Temperatures and Thermal Stresses.

**□** 2. Cooling. Use of He Gas as Cooling Fluid.

**□ 3. Some Technical Solutions.** 

4. Thermal « Shock ».



### **1. Temperatures and Thermal Stresses.**



**O** Stress Sphere/Cylinder at the same dW/dV and R about  $\frac{1}{2}$ .



### **1. Temperatures and Thermal Stresses.**

# **What not to do!**



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 $\Box$  Initial, bell shaped temperature profile created by the beam inside a cylinder ( or sphere) will become uniform by internal, thermal diffusion.

**Time constant** 
$$
\tau_d = \frac{R^2}{8\lambda/c\rho}
$$

 $\Box$  For Cu and W with R=0.5 cm this becomes 30 ms and 50 ms respectively.

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### **1. Temperatures and Thermal Stresses.**

**□ Pulsed Beams: at pulse durations of** ms no cooling occurs: adiabatic heating by  $\Delta T$ . Cooling only after the pulse over time  $t_1$  between pulses.

 $\blacksquare$  T<sub>p</sub> is peak temperature above the temperature of the cooling fluid.

**□ Choose small R!** 

m: masse of sphere c: specific heat of sphere γ: convection coefficient F: surface of sphere

 $T(t) = T_p \times e^{-\frac{t}{\tau}}$ 

$$
Tp = \frac{\Delta T}{1 - e^{\frac{t_1}{\tau}}}
$$

$$
\tau = \frac{mc}{\gamma F}
$$
  Coordinates time  
 constant

$$
\tau = \frac{\rho cR}{3\gamma}
$$
 For spheres

$$
\tau = \frac{\rho cR}{2\gamma}
$$
 For cylinders

The following parameters are relevant for He cooled spheres (and cylinders):

 $\Box$  Diameter of the spheres.

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- $\Box$  He mass flow per free unit cross section between the spheres  $\dot{m}$  (kg.s<sup>-1</sup>.m<sup>-2</sup>).
- $\Box$  Forced cooling convection coefficient γ(W.m<sup>-2</sup>.K<sup>-1</sup>).
- Cooling time constant τ(s).
- $\Box$  Example: He pressure 2 bar; velocity between spheres 40 m/s;diameter of sphere 0.01 m; He mass flow 14.4  $kg.s^{-1}.m^{-2}$ :

 $\blacktriangleright$  y = 2400 W.m<sup>-2</sup>.K<sup>-1</sup>;  $\tau$  = 2 s (longer than internal diffusion time!).

**□** Applicable for a wheel rotating at 30 rpm for ESS.



### **1. Temperatures and Thermal Stresses.**



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### **He Cooled Granular Target for the International Linear Collider-ILC**

- Positrons could be produced in a He-cooled granular W-target, irradiated by an electron beam.
- Maximum energy deposition density: 10 kJ/kg per pulse !!!! (Ref: Robert Chehab). At 5 Hz ( $t_1$  = 200 ms), average power density 50 kW/Kg. Adiabatic temperature rise 7000 K per pulse. Pulse duration  $\rm t_{0}$  = 63 ms.
- Push He cooling to the extreme: spheres ϕ=1mm; He pressure 20 bars; inside velocity 80 m/s: Y = 3.3 x 10<sup>4</sup> W m<sup>-2</sup> K<sup>-1</sup>; cooling time constant
- $\Box$  τ = 13.8 ms.
- **T** Temperature cycling  $T(t)$  (above that of He) can be expressed in a closed form as a function of  $\tau$ ,  $t_0$  and  $t_1$ (see Annex 1).
- Cooling takes place allready during the pulse.  $T(t_0) = 1500$  K. Better than 7000 K, but still high.
- **Q** Rotate Target: Want  $t_0$  gain a dilution by a factor of 20. Need a wheel with a diameter of 8 cm, rotating at 1000 r.p.m.
- $\Box$  The temperature rise in the He during its passage through the target has still to be added. This depends on the He mass flow as well as on the direction it traverses the target: vertically, horizontally or axially.
- $\Box$  Results are shown in Fig.2, ignoring the temperature rise in the He stream.



### **Temperature cycles in the stationary and rotating ILC-target**



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### **Trolling ILC-target**





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> $\Box$  The energy loss per proton passing thin windows ( not much absorption and cascading of secondaries) is in good approximation for all elements:

$$
\frac{dE}{d\zeta} = (1-2)\frac{MeV}{g \cdot cm^{-2}}
$$

$$
\frac{dE}{d\zeta} = (1.6-3.2) \times 10^{-14} \frac{J}{kg \cdot m^{-2}}
$$



### **Beam Windows**

- **Thus the adiabatic temperature rise**  $\Delta T$  **is:**  $\Delta T = \frac{dN}{dt} \times \frac{dE}{d\xi} \times \frac{1}{c}$
- $\Box$  Depends only on specific heat c.
	- **Depends only on specific heat c.**<br>• For the ESS beam with  $\sigma_x = 5$  cm and  $\sigma_y = 1.5$  cm the proton density in the beam center with  $\frac{1}{2\pi \times \sigma_x \sigma_y}$

$$
\frac{dN}{df} = 1.324 \times 10^{17} \text{ protons.m}^{-2}
$$

- Adiabatic max. temperature rises per pulse in Ti and St. Steel are (3.3-6.6) K and (4.2-8.4) K respectively.
- $\Box$  Thermal stresses in the centre are
- $\Box$   $\sigma_{\text{th}}$ =-EαΔT/2, compressive at the centre and decay with 1/r<sup>2</sup> outside the heated zone. They are low, but fatigue will be important!
- $\Box$  Cooling is easy: the cooling time constant for a 0.2 mm thick st.st. window, cooled from only one side with He is about 80 ms, of the same order as the time space between pulses of 50 ms. Thus, temperatures of about  $2 \times \Delta T$  may be expected. However, for a thicknes of 1 mm things get worst.
- $\Box$  How to fight fatigue and radiation damage: Wobbling Window?



### **Beam Windows**

#### □ Better ideas are most welcome.



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■A strong cirular magnetic field inside the target focuses the secondaries inside and along the target. This allows to increase the phase space density (normally forbidden by Lioville's Theorem).



#### **Li-TARGET-LENS**











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- **O** Problems:
- $\Box$  In addition to beam heating: Joule heating!
- **□ For Pi-minus or Anti-Proton production the incident proton** beam is defocused. Its momentum is usually much higher than that of the secondaries?!
- $\Box$  The secondaries are forced to travel along the inside of the target: Re-absorption!
- $\Box$  One must have very good reasons to use pulsed targets. « Only » solution: liquid metal target.



### **Thermal Shock**



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- $\Box$  Instant heating does not move material at time zero, due to its mass inertia! It changes only the equilibrium position of the material: At time zero a bar is too short and it is under stress or pressure.
- $\Box$  The evolution in time thereafter is governed by the wave equation, taking into account the initial conditions and the boundary conditions: at time zero there is uniform stress along the bar, but at both ends of the free bar the stresses remain allways zero.







Fig. 1 Initial axial stress distribution in an instantaneously heated rod.



Fig. 2 The square wave describing the initial axial stress distribution in the rod  $-L \le x \le +L$ .





### **Thermal Shock**



Fig. 4 The time dependence of the temperature in a rod heated by a particle burst of duration  $t_0$ .



The saw-tooth functions travelling in opposite directions and con- $Fig. 5$ stituting the stresses in a rod, heated by a particle burst of duration  $t_0$ .



The development of the axial stress in the rod,  $Fig. 6$ heated by a particle burst of duration  $t_0 = L/4c$ . The time t is measured in units of L/c.



For « slower » heating over τ the stress becomes

$$
\sigma = \frac{\sigma_0 \tau_0}{\tau} = E \alpha \Delta T \times \frac{\tau_0}{\tau}
$$

- $\Box$  τ<sub>o</sub>: Time for sound to traverse half of the target length L/c or the radius of a sphere R/c. c: velocity of sound.
- τ: Pulse duration.
- $\Box$  For CLIC  $\tau$  = 156 ns; R = 1 mm: some shock reduction may be expected???
- $\Box$  Maximum material velocity v can be expressed as a Mach No. :

$$
m = -\frac{1}{c} \alpha \Delta
$$

 $\Box$  Only when m approaches 1, one may speak about thermal shock!

### **Some history**





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Fig. 5 - Overall view of the target assembly. The aluminium container is<br>anodised black to aid cooling via radiation. The hole in the up-<br>stream luminescent screen avoids premature aging and radiation damage of the latter.



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### **Some history**





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## **Some prescribed crimes !!!**



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### **Some prescribed crimes !!!**



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#### **Annex**

## **Temperature cycling with pulsed beams:**

- $\bullet$  Duration of pulses  $t_0$ .
- $\bullet$  Repetition time of pulses  $t_1$ .
- ☛ Cooling constant τ.
- ☛ Adiabatic temperature rise without cooling during one pulse ΔT.

$$
\Box \text{For } 0 < t < t_0: \qquad T_i(t) = \frac{\Delta T \times \tau}{t_0} \times \left(1 - \frac{\left(e^{\frac{-t}{\tau}} / e^{\frac{-t_0}{\tau}}\right) \times \left(e^{\frac{-t_0}{\tau}} - e^{\frac{-t_1}{\tau}}\right)}{1 - e^{\frac{-t_1}{\tau}}}\right)
$$

#### **Annex**



**1** For 
$$
t_0 < t < t_1
$$
 
$$
T_e(t) = \frac{\Delta T \tau}{t_0} \times \left( \frac{e^{-\frac{t}{\tau}}}{e^{-\frac{t_0}{\tau}}} \right) \times \frac{1 - e^{-\frac{t_0}{\tau}}}{1 - e^{-\frac{t_0}{\tau}}}
$$

\n**1** The peak temperature is reached at  $t = t_0$ : 
$$
T(t_0) = \frac{\Delta T \tau}{t_0} \times \frac{1 - e^{-\frac{t_0}{\tau}}}{1 - e^{-\frac{t_0}{\tau}}}
$$

 $\Box$  For short pulses with little cooling during the pulse with  $t_0$  << τ:

$$
T(t_0) = \frac{\Delta T}{1 - e^{-\frac{t_1}{\tau}}}
$$

$$
T(t_1) = \Delta T \times \frac{e^{-\frac{t_1}{\tau}}}{1 - e^{-\frac{t_1}{\tau}}}
$$